

1 **A NEW METHOD FOR CALCULATING CONDUCTION RESPONSE FACTORS FOR**
2 **MULTILAYER CONSTRUCTIONS BASED ON FREQUENCY-DOMAIN SPLINE**
3 **INTERPOLATION (FDSI) AND ASYMPTOTIC ANALYSIS**

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11 **Abstract**

12 Conduction heat transfer through building construction elements is one of the main components of space
13 heating and cooling loads, and, thus, one of the key aspects when planning sustainable energy designs in
14 the building sector. The Response Factors (RF) method sets the base for related dynamic calculations
15 implemented by most well-known Building Energy Simulation (BES) software, and it represents a
16 research topic of present interest. In this regard, this work introduces a new method for calculating
17 conduction Response Factors in building multilayer constructions, based on the definition of an
18 approximated wall model through Frequency-Domain Spline Interpolation (FDSI) and asymptotic
19 analysis. Its conceptual development as well as first validations comparing with existing methods from
20 previous literature are presented. Finally, as a result of applying a table-lookup approach and the
21 possibility of pre-calculating most of the involved operations, an accurate, fast and easy-to-code algorithm
22 is obtained, which constitutes a promising alternative to improve the current state-of-art calculation
23 procedures.

24 **Keywords:** Building transient heat transfer, Thermal Response Factors, spline interpolation, frequency-
25 domain, asymptotic analysis.

26 **Nomenclature**

27 General

28 a, b, c, d coefficients of the piecewise polynomial approximation

29 A_k, B_k, C_k, D_k transmission matrix elements of the k^{th} construction layer

30	A_T, B_T, C_T, D_T	transmission matrix elements of the total multilayer construction
31	c_p	specific heat capacity [$J \cdot kg^{-1} \cdot K^{-1}$]
32	E	Error estimate (%)
33	f_k	cubic spline approximation functions [$W \cdot m^{-2} \cdot K^{-1}$]
34	$F[...]$	Fourier transform
35	$F^{-1}[...]$	inverse Fourier transform
36	G	transfer function
37	h_k	amplitude of the intervals between frequency evaluation points (ω_k) [$rad \cdot s^{-1}$]
38	L	layer thickness [m]
39	j	complex variable
40	MFF	Modified Frequency Function [$W \cdot m^{-2} \cdot K^{-1}$]
41	q	heat flux [$W \cdot m^{-2}$]
42	R	thermal resistance [$m^2 \cdot K \cdot W^{-1}$]
43	r_k	slope of the MFFs at selected frequency evaluation points (ω_k)
44	s	independent variable in the Laplace domain
45	t	time [s]
46	T	temperature [C]
47	\dot{T}	first derivative of the temperature variable [$C \cdot s^{-1}$]
48	T_Δ	shaping function [-]
49	Δt	timestep [s]
50	U	wall's thermal transmittance [$W \cdot m^{-2} \cdot K^{-1}$]
51	X_{RF}, Y_{RF}, Z_{RF}	internal, cross and external terms for the Response Factor method [$W \cdot m^{-2} \cdot K^{-1}$]
52	$X_{CTF}, Y_{CTF}, Z_{CTF}$	internal, cross and external terms for the CTF method [$W \cdot m^{-2} \cdot K^{-1}$]
53	y_k	Value of the MFFs at selected frequency evaluation points (ω_k) [$W \cdot m^{-2} \cdot K^{-1}$]
54	<u><i>Greek symbols</i></u>	
55	α	thermal diffusivity [$m^2 \cdot s^{-1}$]
56	β_k	roots of the transfer function $B_T(s)$
57	λ	thermal conductivity [$W \cdot m^{-1} \cdot K^{-1}$]
58	ψ	amplitude of the frequency characteristic
59	Γ_k, φ_k	auxiliary terms for the recursive calculation of the spline coefficients

60	Φ	weighting coefficients of the previous heat fluxes in the CTF method [-]
61	ρ	material density [$\text{kg}\cdot\text{m}^{-3}$]
62	σ_k	second derivative of the MFFs at selected frequency evaluation points
63	ω	frequency [$\text{rad}\cdot\text{s}^{-1}$]
64	<u>Subscripts</u>	
65	a, b, c, d	relative to the corresponding coefficients of the piecewise polynomial approximation
66	i, k	integer counts
67	A / P	asymptotic / polynomial
68	C / S	cosine / sine
69	H / T	head / tail
70	int / ext	internal / external
71	m	number of timesteps
72	n	total number of elements of a given vector or identifier of a given element into a
73		Response Factor series (X, Y or Z)
74	N	total number of frequency points
75	X, Y, Z	relative to the corresponding term of the Response Factor method
76	<u>Acronyms</u>	
77	BES	Building Energy Simulation
78	CTF	Conductive Transfer Functions
79	DRF	Direct Root Finding
80	FDR	Frequency-Domain Regression
81	FDM	Finite Difference Method
82	FDSI	Frequency-Domain Spline Interpolation
83	FEM	Finite Element Method
84	HVAC	Heating, Ventilating and Air Conditioning
85	RF	Response Factors
86	SSM	State-Space Method
87	<u>Specific notation for FDSI method constants and integration factors</u>	
88	AF	Asymptotic function
89	k	constant (construction-dependent)

90 K integration factor (construction non-dependent) ^(1,2)
 91 ⁽¹⁾ KAHC, KAHS, KATC, KATS, KPCa, KPCb, KPCc, KPCd, KPSa, KPSb, KPSc, KPSd are those
 92 'integration factors' which can be interpreted according to the following criteria:
 93 A/P: asymptotic/polynomial
 94 H/T: head/tail
 95 C/S: cosine/sine
 96 a/b/c/d: associated to the corresponding spline coefficient
 97 ⁽²⁾ KATC0, KATC(+), KATS(+), KATC (-), KATS(-) are particular definitions of the integration factors
 98 to determine the tail asymptotic equivalents

99 1. Introduction

100 As energy and environmental sustainability in the building sector have become increasingly important in
 101 these days, Building Energy Simulation (BES) software has attained a fundamental role in the design of
 102 new constructions and the planning of energy retrofitting actions [1,2]. This software can estimate the
 103 amount of energy required to assure indoor thermal comfort conditions throughout the year (that is to say,
 104 space heating and cooling loads), which allows architects and engineers to better benefit from passive
 105 energy techniques and design more efficient HVAC systems and strategies. In this sense, among other
 106 capabilities, BES software involves methods to evaluate short-wave and long-wave radiative heat
 107 transfer, convective heat flows, one-dimensional heat conduction through multi-layered walls, as well as
 108 the dynamics of the energy facilities within the built environment.

109 In particular, the conduction heat transfer through building construction elements is one of the key
 110 components of space loads. Wang and Chen [3] present an exhaustive review of those methods available
 111 to determine its contribution. Despite the existence of numerical methods [4,5] and the so-called harmonic
 112 or periodic approaches [6,7], currently, the most widely used techniques are the Response Factors (RF)
 113 method and the Conductive Transfer Function (CTF) method, which set the base for the calculations
 114 implemented by well-known BES programs such as Energy-Plus [8] or TRNSYS [9]. These methods are
 115 generally considered to derive from the research conducted by Mitalas and Stephenson [10-12]

116 The Response Factors method calculates the heat flux at discrete times as a function of the previous
 117 temperatures on both sides of the construction (Eqs.1).

$$118 \quad q_{ext}(i \cdot \Delta t) = \sum_{k=0}^{\infty} X_{RF}[k] \cdot T_{ext}[(i - k) \cdot \Delta t] - \sum_{k=0}^{\infty} Y_{RF}[k] \cdot T_{int}[(i - k) \cdot \Delta t] \quad Eq. 1a$$

119
$$q_{int}(i \cdot \Delta t) = \sum_{k=0}^{\infty} Y_{RF}[k] \cdot T_{ext}[(i - k) \cdot \Delta t] - \sum_{k=0}^{\infty} Z_{RF}[k] \cdot T_{int}[(i - k) \cdot \Delta t] \quad Eq. 1b$$

120 T_{int} and T_{ext} strictly are the internal and external surface temperatures. However, for general validation
 121 purposes, numerous case studies from literature often consider them to represent ambient temperatures
 122 including massless inner and outer layers with a thermal resistance value equivalent to that derived from
 123 the corresponding convective heat transfer coefficient (see Tables 5 and 7). The terms $X[k]$, $Y[k]$ and $Z[k]$
 124 for k ranging from zero to infinity are called response factors (RF). These factors tend to zero when k tend
 125 to infinity so, in practice, a finite number of them is accurate enough to describe the construction
 126 dynamics. However, the simulation of HVAC systems integrated in BES programs sometimes requires
 127 time-steps shorter than 1 hour to reproduce realistic control strategies and equipment time responses. In
 128 such situations, the required amount of RF to get good accuracy often becomes inconveniently large for
 129 computer implementation [3, 13].

130 The Conductive Transfer Function method (CTF) [12] reduces the number of terms needed to describe
 131 the construction dynamics. This method expresses the internal and external heat flux values at a given
 132 time in a more convenient form as a function of a finite number of previous temperatures and previous
 133 heat fluxes (see Eqs.2).

134
$$q_{ext}(i \cdot \Delta t) = \sum_{k=0}^N X_{CTF}[k] \cdot T_{ext}[(i - k) \cdot \Delta t] - \sum_{k=0}^N Y_{CTF}[k] \cdot T_{int}[(i - k) \cdot \Delta t] + \sum_{k=1}^M \Phi[k] \cdot q_{ext}[(i - k) \cdot \Delta t] \quad Eq. 2a$$

135
$$q_{int}(i \cdot \Delta t) = \sum_{k=0}^{\infty} Y_{CTF}[k] \cdot T_{ext}[(i - k) \cdot \Delta t] - \sum_{k=0}^{\infty} Z_{CTF}[k] \cdot T_{int}[(i - k) \cdot \Delta t] + \sum_{k=1}^M \Phi[k] \cdot q_{int}[(i - k) \cdot \Delta t] \quad Eq. 2b$$

136 The series of terms $X_{RF}[k]$, $Y_{RF}[k]$ and $Z_{RF}[k]$ for the Response Factor Method (RF) as well as $X_{CTF}[k]$,
 137 $Y_{CTF}[k]$ and $Z_{CTF}[k]$ for the Conductive Transfer Function Method (CTF), can be calculated by several
 138 procedures.

139 The use of Laplace transform methods is probably the most extended one. Briefly, these methods obtain
 140 the heat flux response of the construction to a temperature shaping function (typically a triangle of height
 141 one) in the frequency domain, and then apply the inverse Laplace transform to get the corresponding
 142 solution in the time domain. Eqs.3 show the expressions for the Response Factors, where $A_T(s)$, $B_T(s)$,
 143 $C_T(s)$ and $D_T(s)$ are the terms of the characteristic matrix of the construction (see section 2), while β_k are
 144 the poles of the transfer function $B_T(s)$.

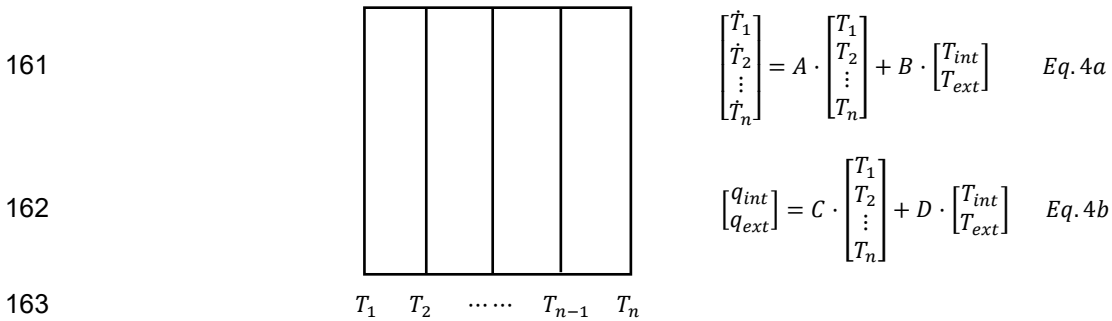
145
$$X_{RF}(t) = t \cdot \left[\frac{D_T(s)}{B_T(s)} \right]_{s=0} + \frac{d}{ds} \left[\frac{D_T(s)}{B_T(s)} \right]_{s=0} + \sum_{k=1}^{\infty} \frac{1}{\beta_k^2} \cdot \left[\frac{D_T(s)}{B_T(s)} \right]_{s=0} \cdot e^{-\beta_k \cdot t} \quad Eq. 3a$$

$$146 \quad Y_{RF}(t) = t \cdot \left[\frac{1}{B_T(s)} \right]_{s=0} + \frac{d}{ds} \left[\frac{1}{B_T(s)} \right]_{s=0} + \sum_{k=1}^{\infty} \frac{1}{\beta_k^2} \cdot \left[\frac{1}{B_T(s)} \right]_{s=0} \cdot e^{-\beta_k t} \quad Eq. 3b$$

$$147 \quad Z_{RF}(t) = t \cdot \left[\frac{A_T(s)}{B_T(s)} \right]_{s=0} + \frac{d}{ds} \left[\frac{A_T(s)}{B_T(s)} \right]_{s=0} + \sum_{k=1}^{\infty} \frac{1}{\beta_k^2} \cdot \left[\frac{A_T(s)}{B_T(s)} \right]_{s=0} \cdot e^{-\beta_k t} \quad Eq. 3c$$

148 This approach was first developed by Stephenson and Mitalas [10-12] and later improved by other
 149 authors [13, 14] in order to avoid pole skipping and enhance calculation efficiency. These methods are
 150 often referred as Direct Root-Finding (DRF) methods, as the inversion process (based on the residue
 151 theorem) involves iterative root finding, which is computationally expensive. In addition, the number of
 152 poles is infinite, so one must settle for a limited number of them. Varela et al. [15] proposed the Direct
 153 Numerical Integration (DNI) of the inversion formula as a viable option.

154 Moreover, other alternatives not based on the Laplace transform have been developed to obtain the RF
 155 or CTF coefficients. Davies [16] evaluated them using elementary time domain solutions for wall heat
 156 flow. Similarly, State-Space Methods (SSM) [17] are also based on a time-domain formulation. As shown
 157 in Figure 1, a temperature node is set for each construction layer. Then, a numeric algorithm (Finite
 158 Element Method - FEM, or Finite Differences Method - FDM) is applied to obtain the terms of the matrices
 159 A, B, C and D of the equivalent space-state model. Once the terms of these matrices are calculated, the
 160 coefficients of the conduction transfer function (CTF) can be deduced by Leverrier's algorithm [18].



164 **Figure 1. General approach of the space-state methods**

165 Seem et al. [19] demonstrated that this approach can also be used to calculate transfer functions for walls
 166 that require two-dimensional models, which could be applied to overcome the limitations of the one-
 167 dimensional approach when dealing with hollow blocks or relevant thermal bridge effects into wall
 168 constructions. In this sense, Kossecka et al. [20] also described a method to derive conduction z-transfer
 169 function coefficients from response factors for 3D wall assemblies. Moreover, Kosny et al. [21] contributed
 170 to the accuracy improvement in whole building thermal modeling tools, through the definition of an
 171 'equivalent wall' concept and the proposal of additional thermal structure factors to account for building
 172 envelope components containing high thermal mass and/or 2D and 3D heat transfer effects.

173 However, state-space formulations for multi-dimensional heat conduction problems often rely on linear
 174 models with high-order matrices involving important computation times. Gao et al. [22] proposed a
 175 solution based on the application of model reduction techniques to decrease computation costs with no
 176 significant accuracy losses.

177 Returning to the Response Factor method for 1D transient heat transfer, another more recent
 178 methodology consists of the Frequency-Domain Regression (FDR) methods developed by Wang et al. [3,
 179 23-26]. They set out a general transfer function whose parameters are estimated by a regression
 180 algorithm (Eq. 5). Starting from several frequency evaluations of the construction dynamics, these
 181 methods find the set of coefficients that minimize the quadratic error between those evaluations and the
 182 ones given by the approximate model.

$$183 \quad G(s) = \frac{\beta_0 + \beta_1 \cdot s + \beta_2 \cdot s^2 + \dots + \beta_{r-1} \cdot s^{r-1} + \beta_r \cdot s^r}{1 + \alpha_1 \cdot s + \alpha_2 \cdot s^2 + \dots + \alpha_{m-1} \cdot s^{m-1} + \alpha_m \cdot s^m} \quad Eq.5$$

184 It is a fast and accurate method. In addition, it allows a direct calculation of the coefficients for the CTF
 185 method instead of the sequence of response factors (RF).

186 Finally, it should be remarked that research on this field is of present interest, what can be supported by
 187 additional improved or innovative approaches contributed in recent years [27-34].

188 Along these lines, this work introduces a new method for calculating conduction response factors (RF) of
 189 building multilayer constructions. It is based on the definition of an approximated wall model through
 190 Frequency-Domain Spline Interpolation (FDSI) and asymptotic analysis. First, temperature evolutions are
 191 expressed as sums of harmonics by means of Fourier transform methods. Then, the FDSI model is
 192 applied to obtain the heat flux solution in the frequency domain. Nevertheless, the particular definition of
 193 such wall model enables to rearrange the Fourier transform inversion integrals so that each response
 194 factor is obtained as the sum of several simple terms. In addition, those terms depending on the
 195 construction thermo-physical properties can be separated and pre-calculated, making it possible to use
 196 an efficient table-lookup approach. Therefore, the FDSI method provides an accurate, fast and easy-to-
 197 code alternative to the current RF calculation methodologies.

198 **2. Frequency response in multilayer constructions**

199 Given the Laplace model of a construction, it is easy to evaluate the gain and the phase shift associated
 200 to each frequency. For a particular layer (k), its mathematical expression is the following [3]:

$$201 \quad \begin{bmatrix} T_{ext} \\ q_{ext} \end{bmatrix} = \begin{bmatrix} A_k(s) & B_k(s) \\ C_k(s) & D_k(s) \end{bmatrix} \cdot \begin{bmatrix} T_{int} \\ q_{int} \end{bmatrix} \xrightarrow{s=j\omega} \begin{bmatrix} T_{ext} \\ q_{ext} \end{bmatrix} = \begin{bmatrix} A_k(j\omega) & B_k(j\omega) \\ C_k(j\omega) & D_k(j\omega) \end{bmatrix} \cdot \begin{bmatrix} T_{int} \\ q_{int} \end{bmatrix} \quad Eq.6$$

202
$$\begin{bmatrix} A_k(j\omega) & B_k(j\omega) \\ C_k(j\omega) & D_k(j\omega) \end{bmatrix} = \begin{bmatrix} \cosh\left(L_k \cdot \sqrt{\frac{j\omega}{\alpha_k}}\right) & \frac{\sinh\left(L_k \cdot \sqrt{\frac{j\omega}{\alpha_k}}\right)}{\lambda_k \cdot \sqrt{\frac{j\omega}{\alpha_k}}} \\ \lambda_k \cdot \sqrt{\frac{j\omega}{\alpha_k}} \cdot \sinh\left(L_k \cdot \sqrt{\frac{j\omega}{\alpha_k}}\right) & \cosh\left(L_k \cdot \sqrt{\frac{j\omega}{\alpha_k}}\right) \end{bmatrix} \quad \text{Eq. 7}$$

203 Numeric evaluation of the matrix above for each frequency yields four complex numbers that describe the
 204 gain and phase shift of the layer thermal response associated to these frequencies. Besides, the
 205 characteristic matrix of the whole construction can be expressed as the product of the matrices for each
 206 individual layer:

207
$$\begin{bmatrix} A_T(j\omega) & B_T(j\omega) \\ C_T(j\omega) & D_T(j\omega) \end{bmatrix} = \begin{bmatrix} A_1(j\omega) & B_1(j\omega) \\ C_1(j\omega) & D_1(j\omega) \end{bmatrix} \cdot \begin{bmatrix} A_2(j\omega) & B_2(j\omega) \\ C_2(j\omega) & D_2(j\omega) \end{bmatrix} \cdots \begin{bmatrix} A_{n-1}(j\omega) & B_{n-1}(j\omega) \\ C_{n-1}(j\omega) & D_{n-1}(j\omega) \end{bmatrix} \cdot \begin{bmatrix} A_n(j\omega) & B_n(j\omega) \\ C_n(j\omega) & D_n(j\omega) \end{bmatrix} \quad \text{Eq. 8}$$

208 If heat fluxes are written down as a function of temperatures, the following formulation (Eq.9) is obtained.

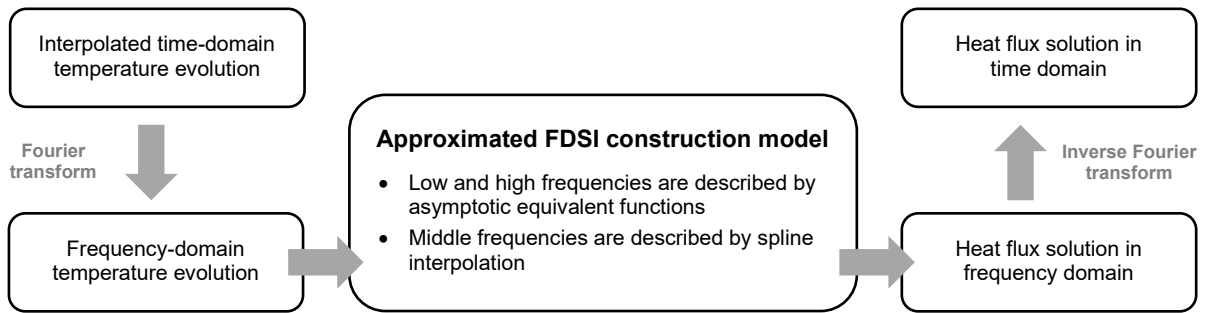
209
$$\begin{bmatrix} q_{int} \\ q_{ext} \end{bmatrix} = \begin{bmatrix} \frac{D_T(j\omega)}{B_T(j\omega)} & -\frac{1}{B_T(j\omega)} \\ 1 & -\frac{A_T(j\omega)}{B_T(j\omega)} \end{bmatrix} \cdot \begin{bmatrix} T_{ext} \\ T_{int} \end{bmatrix} = \begin{bmatrix} X(\omega) & -Y(\omega) \\ Y(\omega) & -Z(\omega) \end{bmatrix} \cdot \begin{bmatrix} T_{ext} \\ T_{int} \end{bmatrix} \quad \text{Eq. 9}$$

210 Therefore:

211
$$X(\omega) = \frac{D_T(j\omega)}{B_T(j\omega)}; \quad Y(\omega) = \frac{1}{B_T(j\omega)}; \quad Z(\omega) = \frac{A_T(j\omega)}{B_T(j\omega)} \quad \text{Eqs. 10}$$

212 3. Description of the Frequency-Domain Spline Interpolation (FDSI) algorithm

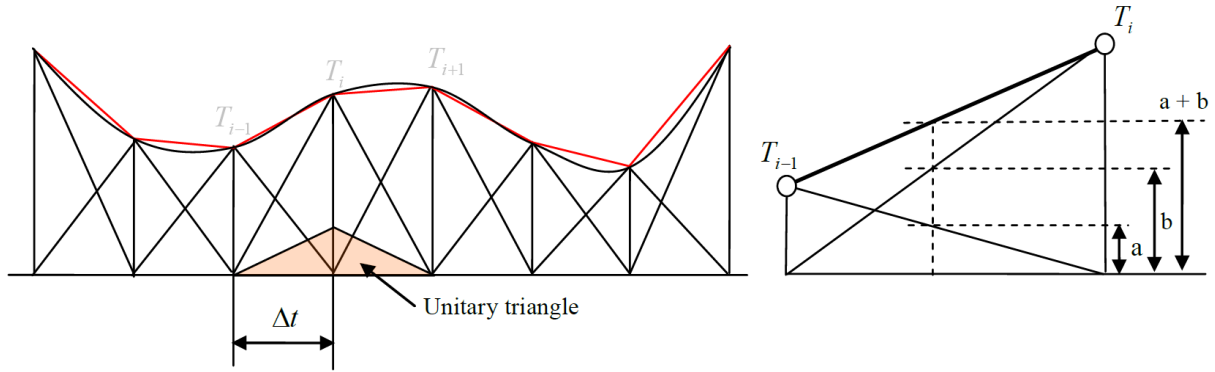
213 The basic idea of the FDSI method is to split the temperature evolution into a sum of harmonics using the
 214 Fourier transform. Then, by means of an approximated model for the construction, the gain and phase
 215 shift for each frequency are calculated. Finally, this method gets the heat fluxes in time domain using the
 216 inverse Fourier transform.



217
 218 **Figure 2. Block diagram for the FDSI fundamental conception**

219 In order to simplify this method, the temperature evolution between sampling points is often considered
 220 as linear. This is a reasonable assumption when such evolution is sufficiently slow compared to the time
 221 interval between these points, but not when there are abrupt temperature changes.

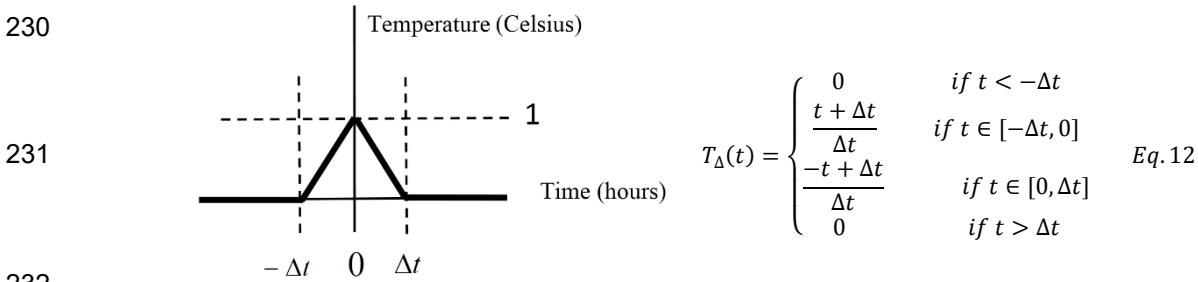
222 Considering this hypothesis, the interpolated time-domain function can be split into a set of triangular
 223 pulses, as it is represented in Figure 3.



224
 225 **Figure 3. Breakdown of a linearly interpolated temperature evolution into unitary triangles**

226 Therefore, the temperature function (for both the external or the internal temperature variables) can be
 227 expressed as the sum of a set of scaled and shifted unitary triangles (T_{Δ}) which constitute the so-called
 228 'shaping function'. Figure 4 describes in detail the characteristics of such function.

229
$$T(t) = \sum_{k=-\infty}^m T(k \cdot \Delta t) \cdot T_{\Delta}(t - k \cdot \Delta t) \quad \text{with } m \cdot \Delta t \geq t \quad \text{Eq. 11}$$



233 **Figure 4. Description of a triangular shaping function**

234 Shaping functions might also adopt expressions other than triangular, thus describing different types of
 235 interpolation [34]. In this case, starting from the temperature as a linear combination of shaping functions,
 236 the Fourier transforms yields an equivalent sum of harmonics. For the external temperature evolution this
 237 can be expressed through the equation Eq.13a.

238
$$T_{ext}(\omega) = F[T_{ext}(t)] = F \left[\sum_{k=-\infty}^m T_{ext}(k \cdot \Delta t) \cdot T_{\Delta}(t - k \cdot \Delta t) \right] = \sum_{k=-\infty}^m T_{ext}(k \cdot \Delta t) \cdot F[T_{\Delta}(t - k \cdot \Delta t)]$$

239
$$= \sum_{k=-\infty}^m T_{ext}(k \cdot \Delta t) \cdot e^{-k \cdot \Delta t \cdot j\omega} \cdot F[T_{\Delta}(t)] \quad \text{with } m \cdot \Delta t \geq t \quad \text{Eq. 13a}$$

240 Similarly, Eq.13b can be derived for the internal temperature evolution:

241
$$T_{int}(\omega) = F[T_{int}(t)] = \sum_{k=-\infty}^m T_{int}(k \cdot \Delta t) \cdot e^{-k \cdot \Delta t \cdot j\omega} \cdot F[T_{\Delta}(t)] \quad \text{with } m \cdot \Delta t \geq t \quad \text{Eq. 13b}$$

242 According to this formulation, the heat flux solutions in the frequency domain are obtained as follows:

243
$$q_{ext}(\omega) = X(\omega) \cdot T_{ext}(\omega) - Y(\omega) \cdot T_{int}(\omega) =$$

244
$$= X(\omega) \cdot \sum_{k=-\infty}^m T_{ext}(k \cdot \Delta t) \cdot e^{-k \cdot \Delta t \cdot j\omega} \cdot F[T_{\Delta}(t)] - Y(\omega) \cdot \sum_{k=-\infty}^m T_{int}(k \cdot \Delta t) \cdot e^{-k \cdot \Delta t \cdot j\omega} \cdot F[T_{\Delta}(t)] \quad \text{Eq. 14a}$$

245
$$q_{int}(\omega) = Y(\omega) \cdot T_{ext}(\omega) - Z(\omega) \cdot T_{int}(\omega) =$$

246
$$= Y(\omega) \cdot \sum_{k=-\infty}^m T_{ext}(k \cdot \Delta t) \cdot e^{-k \cdot \Delta t \cdot j\omega} \cdot F[T_{\Delta}(t)] - Z(\omega) \cdot \sum_{k=-\infty}^m T_{int}(k \cdot \Delta t) \cdot e^{-k \cdot \Delta t \cdot j\omega} \cdot F[T_{\Delta}(t)] \quad \text{Eq. 14b}$$

247 At this point, heat fluxes in time domain can be calculated via the inverse Fourier transform.

248
$$q_{ext}(t) = F^{-1}[q_{ext}(\omega)]; \quad q_{int}(t) = F^{-1}[q_{int}(\omega)] \quad \text{Eq. 15}$$

249
$$q_{ext}(t) = \sum_{k=-\infty}^m T_{ext}(k \cdot \Delta t) \cdot F^{-1}[X(\omega) \cdot e^{-k \cdot \Delta t \cdot j\omega} \cdot F[T_{\Delta}(t)]] - \sum_{k=-\infty}^m T_{int}(k \cdot \Delta t) \cdot F^{-1}[Y(\omega) \cdot e^{-k \cdot \Delta t \cdot j\omega} \cdot F[T_{\Delta}(t)]] =$$

250
$$= \sum_{k=-\infty}^m T_{ext}(k \cdot \Delta t) \cdot F^{-1}[X(\omega) \cdot F[T_{\Delta}(t)]](t - k \cdot \Delta t) - \sum_{k=-\infty}^m T_{int}(k \cdot \Delta t) \cdot F^{-1}[Y(\omega) \cdot F[T_{\Delta}(t)]](t - k \cdot \Delta t) \quad \text{Eq. 16a}$$

251
$$q_{int}(t) = \sum_{k=-\infty}^m T_{ext}(k \cdot \Delta t) \cdot F^{-1}[Y(\omega) \cdot F[T_{\Delta}(t)]](t - k \cdot \Delta t) - \sum_{k=-\infty}^m T_{int}(k \cdot \Delta t) \cdot F^{-1}[Z(\omega) \cdot F[T_{\Delta}(t)]](t - k \cdot \Delta t) \quad \text{Eq. 16b}$$

252 In the formulation above, the following expression yields the response factors by conveniently substituting

253 in Eq.17 the term RF (Response Factor) by each corresponding series of factors (X, Y or Z).

254
$$RF(t) = F^{-1}[RF(\omega) \cdot F[T_{\Delta}(t)]] =$$

255
$$= \frac{1}{\sqrt{2\pi}} \int_{-\infty}^{\infty} |RF(\omega)| \cdot e^{j \cdot \text{arg}[RF(\omega)]} \cdot F[T_{\Delta}(t)] \cdot e^{j\omega t} d\omega = \frac{1}{\sqrt{2\pi}} \int_{-\infty}^{\infty} |RF(\omega)| \cdot e^{(\omega t + \text{arg}[RF(\omega)]) \cdot j} \cdot F[T_{\Delta}(t)] d\omega \quad \text{Eq. 17}$$

256 If the shaping function is even (as it happens to be when linear interpolation is applied to the temperature

257 evolution) the above integral can be rewritten in a much simpler way.

258
$$RF(t) = F^{-1}[RF(\omega) \cdot F[T_{\Delta}(t)]] = \sqrt{\frac{2}{\pi}} \int_{-\infty}^{\infty} |RF(\omega)| \cdot F[T_{\Delta}(t)] \cdot \cos(\omega t + \text{arg}[RF(\omega)]) \cdot d\omega \quad \text{Eq. 18}$$

259
$$RF(t) = F^{-1}[RF(\omega) \cdot F[T_{\Delta}(t)]] =$$

260
$$= \sqrt{\frac{2}{\pi}} \int_{-\infty}^{\infty} |RF(\omega)| \cdot F[T_{\Delta}(t)] \cdot [\cos(\omega t) \cdot \cos(\text{arg}[RF(\omega)]) - \sin(\omega t) \cdot \sin(\text{arg}[RF(\omega)])] \cdot d\omega \quad \text{Eq. 19}$$

261 These expressions could be solved analytically if the exact phase shift and the amplitude gain functions

262 were used for each frequency. However, this becomes exceedingly complex, so X(ω), Y(ω) and Z(ω) will

263 be substituted by an approximate description inferred from a limited number of frequency evaluations.

264 With this idea in perspective, it results useful to group all the terms that depend on the construction into
 265 separate functions which, from now on, will be called “Modified Frequency Functions” or MFFs. Eqs.20
 266 develop this approach for each response factor.

$$267 \quad X_{RF}(t) = \sqrt{\frac{2}{\pi}} \int_{-\infty}^{\infty} MFF_{XC}(\omega) \cdot F[T_{\Delta}(t)] \cdot \cos(\omega t) \cdot d\omega - \sqrt{\frac{2}{\pi}} \int_{-\infty}^{\infty} MFF_{XS}(\omega) \cdot F[T_{\Delta}(t)] \cdot \sin(\omega t) \cdot d\omega \quad Eq. 20a$$

$$268 \quad Y_{RF}(t) = \sqrt{\frac{2}{\pi}} \int_{-\infty}^{\infty} MFF_{YC}(\omega) \cdot F[T_{\Delta}(t)] \cdot \cos(\omega t) \cdot d\omega - \sqrt{\frac{2}{\pi}} \int_{-\infty}^{\infty} MFF_{YS}(\omega) \cdot F[T_{\Delta}(t)] \cdot \sin(\omega t) \cdot d\omega \quad Eq. 20b$$

$$269 \quad Z_{RF}(t) = \sqrt{\frac{2}{\pi}} \int_{-\infty}^{\infty} MFF_{ZC}(\omega) \cdot F[T_{\Delta}(t)] \cdot \cos(\omega t) \cdot d\omega - \sqrt{\frac{2}{\pi}} \int_{-\infty}^{\infty} MFF_{ZS}(\omega) \cdot F[T_{\Delta}(t)] \cdot \sin(\omega t) \cdot d\omega \quad Eq. 20c$$

270 where:

$$271 \quad MFF_{XC}(\omega) = |X(\omega)| \cdot \cos[\arg[X(\omega)]] \quad Eq. 21a$$

$$272 \quad MFF_{XS}(\omega) = |X(\omega)| \cdot \sin[\arg[X(\omega)]] \quad Eq. 21b$$

$$273 \quad MFF_{YC}(\omega) = |Y(\omega)| \cdot \cos[\arg[Y(\omega)]] \quad Eq. 21c$$

$$274 \quad MFF_{YS}(\omega) = |Y(\omega)| \cdot \sin[\arg[Y(\omega)]] \quad Eq. 21d$$

$$275 \quad MFF_{ZC}(\omega) = |Z(\omega)| \cdot \cos[\arg[Z(\omega)]] \quad Eq. 21e$$

$$276 \quad MFF_{ZS}(\omega) = |Z(\omega)| \cdot \sin[\arg[Z(\omega)]] \quad Eq. 21f$$

277 If MFFs are replaced by third order piecewise polynomials inside a given frequency range $[\omega_1, \omega_2]$ and by
 278 asymptotic functions outside this range, the following general expression (Eq.22) for the X response
 279 factor is obtained.

$$280 \quad X_{RF}(n \cdot \Delta t) = \sqrt{\frac{2}{\pi}} \cdot \left\{ \begin{array}{l} \int_0^{\omega_1} [k_{XHC1} \cdot AF_{HC1}(\omega) + k_{XHC2} \cdot AF_{HC2}(\omega)] \cdot F[T_{\Delta}(t)] \cdot \cos(\omega \cdot n \cdot \Delta t) \cdot d\omega + \\ + \sum_{k=1}^{N-1} \left[\int_{\omega_i}^{\omega_{i+1}} (a_{XC(k)} \cdot \omega^3 + b_{XC(k)} \cdot \omega^2 + c_{XC(k)} \cdot \omega + d_{XC(k)}) \cdot F[T_{\Delta}(t)] \cdot \cos(\omega \cdot n \cdot \Delta t) \cdot d\omega \right] + \\ + \int_{\omega_N}^{\infty} [k_{XTC1} \cdot AF_{TC1}(\omega) + k_{XTC2} \cdot AF_{TC2}(\omega)] \cdot F[T_{\Delta}(t)] \cdot \cos(\omega \cdot n \cdot \Delta t) \cdot d\omega - \\ - \int_0^{\omega_1} [k_{XHS1} \cdot AF_{HS1}(\omega) + k_{XHS2} \cdot AF_{HS2}(\omega)] \cdot F[T_{\Delta}(t)] \cdot \sin(\omega \cdot n \cdot \Delta t) \cdot d\omega - \\ - \sum_{k=1}^{N-1} \left[\int_{\omega_i}^{\omega_{i+1}} (a_{XS(k)} \cdot \omega^3 + b_{XS(k)} \cdot \omega^2 + c_{XS(k)} \cdot \omega + d_{XS(k)}) \cdot F[T_{\Delta}(t)] \cdot \sin(\omega \cdot n \cdot \Delta t) \cdot d\omega \right] - \\ - \int_0^{\omega_1} [k_{XTS1} \cdot AF_{TS1}(\omega) + k_{XTS2} \cdot AF_{TS2}(\omega)] \cdot F[T_{\Delta}(t)] \cdot \sin(\omega \cdot n \cdot \Delta t) \cdot d\omega \end{array} \right\} \quad Eq. 22$$

281 Eq.22 can be adapted in the same way for Y and Z response factors, but they are not presented here for
 282 the sake of simplicity.

283 It should be noted that the asymptotic approximations of the MFFs out of the selected frequency range
 284 have been defined through the following general expression:

285 $MFF(\omega) \approx k_1 \cdot AF_1(\omega) + k_2 \cdot AF_2(\omega)$ Eq. 23

286 where k_1 and k_2 are constants, and $AF_1(\omega)$ and $AF_2(\omega)$ are functions that describe the asymptotic
 287 behavior of the MFFs along the ‘head’ $[0, \omega_1]$ and ‘tail’ $[\omega_2, \infty)$ frequency intervals. Sections 4 and 6
 288 describe in detail the particular expressions of these constants and functions that should be applied on
 289 the calculation of each response factor (X, Y, or Z) along the head or tail intervals.

290 Finally, Eq.22 can be rearranged so that each response factor is simply obtained as the sum of several
 291 terms. Each addend will be the product of a factor that depends on the construction characteristics
 292 (lowercase constants) by another factor that can be pre-calculated (uppercase constants). Next, Eq.24
 293 shows the expression of the X response factor as an example, but, again, similar expressions can be
 294 derived in the same way for Y and Z factors.

295 $X_{RF}(n \cdot \Delta t) =$

$$296 = \sqrt{\frac{2}{\pi}} \cdot \left\{ \begin{aligned} & k_{XHC1} \cdot KAHC1_n + k_{XHC2} \cdot KAHC2_n + \sum_{k=1}^{N-1} [a_{XC(k)} \cdot KPCa_{n,k} + b_{XC(k)} \cdot KPCb_{n,k} + c_{XC(k)} \cdot KPCC_{n,k} + d_{XC(k)} \cdot KPCC_{n,k}] + \\ & + k_{XHS1} \cdot KAHS1_n + k_{XHS2} \cdot KAHS2_n - k_{XTC1} \cdot KATC1_n - k_{XTC2} \cdot KATC2_n - \\ & - \sum_{k=1}^{N-1} [a_{XS(k)} \cdot KPSa_{n,k} + b_{XS(k)} \cdot KPSb_{n,k} + c_{XS(k)} \cdot KPSC_{n,k} + d_{XS(k)} \cdot KPSC_{n,k}] - k_{XTS1} \cdot KATS1_n - k_{XTS2} \cdot KATS2_n \end{aligned} \right\} \text{Eq. 24}$$

297 The uppercase constants (which from now on will be named as “integration factors”) do not depend on
 298 the construction and can be obtained analytically. Therefore, they need to be calculated only once and
 299 then can be embedded as part of the FDSI main algorithm developed to obtain the response factors.
 300 Section 4 presents the detailed expressions of these integration factors.

301 On the other hand, the constants that depend on the construction characteristics are the spline
 302 coefficients for each frequency interval and the constants associated to the definition of the asymptotic
 303 equivalents. All of them can be easily calculated by evaluating the MFFs at several frequency points, as it
 304 is described in sections 5 and 6.

305 In conclusion, once all the integrals are pre-calculated and stored inside a large table, the operations to
 306 be performed by the FDSI method are reduced to the following:

- 307 • Evaluation of the Modified Frequency Functions (MFFs) at several frequency points, logarithmically
 308 spaced.
- 309 • Spline interpolation of the MFFs.
- 310 • Calculation of the coefficients associated to the asymptotic equivalents.
- 311 • Sum of the products of the spline coefficients by the integration terms to calculate each response
 312 factor.

313

314 **4. Generation of the integration factors**

315 In order to generate the table of terms to be embedded within the method as pre-calculated factors, it is
 316 necessary to solve a certain number of definite integrals. As it has been previously mentioned, there are
 317 two types of integration factors, which can be referred as polynomial and asymptotic integration factors.

318 The polynomial factors are those derived from the piecewise polynomial interpolation that approximate
 319 MFFs at middle frequencies. The asymptotic factors derive from the definition of equivalent asymptotic
 320 functions that approximate MFFs at high and low frequencies.

321 **4.1. Integration factors for the spline coefficients that approximate MFFs at middle frequencies.**

322 Table 1 gathers the integration factors that multiply the spline coefficients at each frequency interval (see
 323 Eq.24). There are $8 \cdot (k-1)$ terms of this kind for each response factor, being k the number of frequency
 324 points where the MFFs are evaluated.

325 Table 1: Integration factors for the spline interpolation

Cosine factors	Sine factors
$KPCa_{n,k} = \int_{\omega^{(k-1)}}^{\omega^{(k)}} \omega^3 \cdot F[T_{\Delta}(t)] \cdot \cos(\omega \cdot n \cdot \Delta t) \cdot d\omega$	$KPSa_{n,k} = \int_{\omega^{(k-1)}}^{\omega^{(k)}} \omega^3 \cdot F[T_{\Delta}(t)] \cdot \sin(\omega \cdot n \cdot \Delta t) \cdot d\omega$
$KPCb_{n,k} = \int_{\omega^{(k-1)}}^{\omega^{(k)}} \omega^2 \cdot F[T_{\Delta}(t)] \cdot \cos(\omega \cdot n \cdot \Delta t) \cdot d\omega$	$KPSb_{n,k} = \int_{\omega^{(k-1)}}^{\omega^{(k)}} \omega^2 \cdot F[T_{\Delta}(t)] \cdot \sin(\omega \cdot n \cdot \Delta t) \cdot d\omega$
$KPCc_{n,k} = \int_{\omega^{(k-1)}}^{\omega^{(k)}} \omega \cdot F[T_{\Delta}(t)] \cdot \cos(\omega \cdot n \cdot \Delta t) \cdot d\omega$	$KPSc_{n,k} = \int_{\omega^{(k-1)}}^{\omega^{(k)}} \omega \cdot F[T_{\Delta}(t)] \cdot \sin(\omega \cdot n \cdot \Delta t) \cdot d\omega$
$KPCd_{n,k} = \int_{\omega^{(k-1)}}^{\omega^{(k)}} F[T_{\Delta}(t)] \cdot \cos(\omega \cdot n \cdot \Delta t) \cdot d\omega$	$KPSd_{n,k} = \int_{\omega^{(k-1)}}^{\omega^{(k)}} F[T_{\Delta}(t)] \cdot \sin(\omega \cdot n \cdot \Delta t) \cdot d\omega$

326 (*) $F[T_{\Delta}(t)] = \frac{1-\cos(\omega \cdot \Delta t)}{\omega^2 \cdot \Delta t}$ for linear interpolation.

327 **4.2. Integration factors for the asymptotic functions that approximate MFFs at low and high**
 328 **frequencies.**

329 So far, eight different terms of this kind have been proposed in a general description of the asymptotic
 330 equivalents (see Eq.24): KAHC1, KAHC2, KAHS1 and KAHS2 for the low-frequency or head interval, and
 331 KATC1, KATC2, KATS1 and KATS2 for the high-frequency or tail interval.

332 The low-frequency asymptotic behavior can be described by the same four head functions for any series
 333 of response factors (X, Y and Z) and in any particular case. However, that is not the case for the tail
 334 terms. Fortunately, it can be proved that proper combinations of only 5 different integration factors are
 335 needed to characterize this high-frequency behavior in any situation. From now on, these tail integration

336 factors are named as $KATC0$, $KATC(+)$, $KATC(-)$, $KATS(+)$ and $KATS(-)$. The relation between them and
 337 the general asymptotic tail terms showed in Eq.24 depends on the response factor being calculated, as
 338 well as on the existence or inexistence of a zero-inertia outermost or innermost wall layer.

339 The rearrangement of these integration factors is a consequence of the particular form of the MFFs.
 340 Further considerations for practical implementation of the FDSI method are given in Appendix A.
 341 Nevertheless, a complete derivation of the asymptotic analysis that leads to these terms would be too
 342 long to be presented on this paper and has been intentionally omitted here.

343 Next, Table 2 compiles the aforementioned integration factors, which multiply the construction-dependent
 344 constants associated to the definition of the asymptotic equivalents. In conclusion, one can observe that 8
 345 integration asymptotic factors need to be calculated.

346 Table 2: Integration factors for the approximate asymptotic functions

Head asymptotic equivalents (low frequency)	Tail asymptotic equivalents (high frequency)
$KAHC1_n = \int_0^{\omega_1} F[T_\Delta(t)] \cdot \cos(\omega \cdot n \cdot \Delta t) \cdot d\omega$	$KATC0_n = \int_{\omega_N}^{\infty} F[T_\Delta(t)] \cdot \cos(\omega \cdot n \cdot \Delta t) \cdot d\omega$
$KAHC2_n = \int_0^{\omega_1} \omega \cdot (e^\omega - 1) \cdot F[T_\Delta(t)] \cdot \cos(\omega \cdot n \cdot \Delta t) \cdot d\omega$	$KATC(+)_n = \int_{\omega_N}^{\infty} \omega^{0.5} \cdot F[T_\Delta(t)] \cdot \cos(\omega \cdot n \cdot \Delta t) \cdot d\omega$
$KAHS1_n = \int_0^{\omega_1} (e^\omega - 1) \cdot F[T_\Delta(t)] \cdot \sin(\omega \cdot n \cdot \Delta t) \cdot d\omega$	$KATS(+)_n = \int_{\omega_N}^{\infty} \omega^{0.5} \cdot F[T_\Delta(t)] \cdot \sin(\omega \cdot n \cdot \Delta t) \cdot d\omega$
$KAHS2_n = 0$	$KATC(-)_n = \int_{\omega_N}^{\infty} \omega^{-0.5} \cdot F[T_\Delta(t)] \cdot \cos(\omega \cdot n \cdot \Delta t) \cdot d\omega$
	$KATS(-)_n = \int_{\omega_N}^{\infty} \omega^{-0.5} \cdot F[T_\Delta(t)] \cdot \sin(\omega \cdot n \cdot \Delta t) \cdot d\omega$

347
 348 Finally, it should be noted that both types of integration factors must be stored into a large table.
 349 However, this is not an issue for modern computers in order to handle the FDSI method easily. For
 350 example, if the number of intervals between frequency evaluations ($k-1$) is 1024 and the number of
 351 response factors is 300, the total number of terms to be handled is the following:

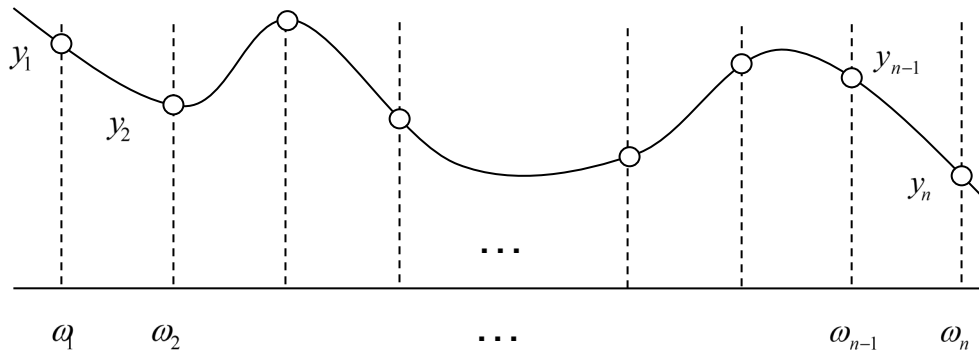
$$352 \quad m = 8 \cdot (k-1) \cdot n + 8 \cdot n \approx 8 \cdot 1024 \cdot 300 = 2457600 \text{ terms} \quad \text{Eq.25}$$

353 If each element of the table is stored as a double precision floating point number (8 bytes), the total size
 354 of the table would be 19660800 bytes, that is to say, 18.75 MB. Modern computers can easily hold tables
 355 this size and larger in RAM memory. Similar results can be obtained with less frequency evaluations and,
 356 therefore, with a smaller factor table, as it will be shown in the validation section.

357

358 5. Spline interpolation of the Modified Frequency Functions (MFFs)

359 An approximate model for the Modified Frequency Functions (MFFs) at middle frequencies can be
360 performed by piecewise polynomials (see Figure 5). The target is to have an ensemble of simple
361 expressions that describes the MFFs between the frequency evaluation points with sufficient accuracy.



362

363 **Figure 5. Piecewise polynomial interpolation approach**

364 In order to achieve this goal, the FDSI method turns to cubic splines, as they are suitable for continuous
365 smooth functions. Starting from a set of n evaluations of the MFFs at logarithmically-spaced frequencies
366 within the selected range $[\omega_1, \omega_n]$, (n-1) cubic polynomials (f_k) are defined, and their corresponding
367 coefficients a_k, b_k, c_k and d_k (Eq.26) are calculated.

368
$$f_k(\omega) = a_k \cdot \omega^3 + b_k \cdot \omega^2 + c_k \cdot \omega + d_k \quad Eq.26$$

369 This calculation can be performed by a linear-time complexity algorithm. Particularly, based on the
370 application of not-a-knot boundary conditions, Appendix B presents a detailed description of the recursive
371 calculation procedure to get the spline coefficients in global coordinates.

372

373 6. Asymptotic analysis of the Modified Frequency Functions (MFFs)

374 Spline interpolation is able to reproduce with excellent precision the so-called Modified Frequency
375 Functions (MFFs) inside a given frequency range. However, outside this range the error becomes
376 unacceptably large since splines are not suitable for extrapolation. Luckily, when the frequency is
377 sufficiently low or sufficiently high, it is possible to replace the complex dynamics of the construction by
378 simple asymptotic equivalents that can be easily integrated into the method.

379 Given a general one-layer construction, the corresponding asymptotic equivalents can be expressed as
380 follows in Table 3.

381 Table 3: Expression of the asymptotic equivalents for a one-layer construction

Term	Exact Modified Frequency Functions (one-layer construction)	Approximated functions
X	$ X(\omega) \cdot \cos\varphi(\omega) = \left \frac{D_T(j\omega)}{B_T(j\omega)} \right \cdot \cos \left(\arg \left(\frac{D_T(j\omega)}{B_T(j\omega)} \right) \right) =$ $= \left \lambda \cdot \sqrt{\frac{j\omega}{\alpha}} \cdot \coth \left(L \cdot \sqrt{\frac{j\omega}{\alpha}} \right) \right \cdot \cos \left(\arg \left[\lambda \cdot \sqrt{\frac{j\omega}{\alpha}} \cdot \coth \left(L \cdot \sqrt{\frac{j\omega}{\alpha}} \right) \right] \right)$	<p>At high frequencies</p> $ X(\omega) \cdot \cos\varphi(\omega) \approx \frac{\lambda}{\sqrt{2\alpha}} \cdot \omega^{0.5}$ <p>At low frequencies</p> $ X(\omega) \cdot \cos\varphi(\omega) \approx \frac{\lambda}{L} + \frac{\lambda \cdot L^3 \cdot \omega \cdot (e^\omega - 1)}{45 \cdot \alpha^2}$
	$ X(\omega) \cdot \sin\varphi(\omega) = \left \frac{D_T(j\omega)}{B_T(j\omega)} \right \cdot \sin \left(\arg \left(\frac{D_T(j\omega)}{B_T(j\omega)} \right) \right) =$ $= \left \lambda \cdot \sqrt{\frac{j\omega}{\alpha}} \cdot \coth \left(L \cdot \sqrt{\frac{j\omega}{\alpha}} \right) \right \cdot \sin \left(\arg \left[\lambda \cdot \sqrt{\frac{j\omega}{\alpha}} \cdot \coth \left(L \cdot \sqrt{\frac{j\omega}{\alpha}} \right) \right] \right)$	<p>At high frequencies</p> $ X(\omega) \cdot \sin\varphi(\omega) \approx \frac{\lambda}{\sqrt{2\alpha}} \cdot \omega^{0.5}$ <p>At low frequencies</p> $ X(\omega) \cdot \sin\varphi(\omega) \approx \frac{\lambda \cdot L \cdot (e^\omega - 1)}{3\alpha}$
Y	$ Y(\omega) \cdot \cos\varphi(\omega) = \left \frac{1}{B_T(j\omega)} \right \cdot \cos \left(\arg \left(\frac{1}{B_T(j\omega)} \right) \right) =$ $= \left \frac{\lambda \cdot \sqrt{\frac{j\omega}{\alpha}}}{\sinh \left(L \cdot \sqrt{\frac{j\omega}{\alpha}} \right)} \right \cdot \cos \left(\arg \left[\frac{\lambda \cdot \sqrt{\frac{j\omega}{\alpha}}}{\sinh \left(L \cdot \sqrt{\frac{j\omega}{\alpha}} \right)} \right] \right)$	<p>At high frequencies</p> $ Y(\omega) \cdot \cos\varphi(\omega) \approx 0$ <p>At low frequencies</p> $ Y(\omega) \cdot \cos\varphi(\omega) \approx \frac{\lambda}{L} + \frac{7k \cdot L^3 \cdot \omega \cdot (e^\omega - 1)}{360 \cdot \alpha^2}$
	$ Y(\omega) \cdot \sin\varphi(\omega) = \left \frac{1}{B_T(j\omega)} \right \cdot \sin \left(\arg \left(\frac{1}{B_T(j\omega)} \right) \right) =$ $= \left \frac{\lambda \cdot \sqrt{\frac{j\omega}{\alpha}}}{\sinh \left(L \cdot \sqrt{\frac{j\omega}{\alpha}} \right)} \right \cdot \sin \left(\arg \left[\frac{\lambda \cdot \sqrt{\frac{j\omega}{\alpha}}}{\sinh \left(L \cdot \sqrt{\frac{j\omega}{\alpha}} \right)} \right] \right)$	<p>At high frequencies</p> $ Y(\omega) \cdot \sin\varphi(\omega) \approx 0$ <p>At low frequencies</p> $ Y(\omega) \cdot \sin\varphi(\omega) \approx \frac{\lambda \cdot L \cdot (e^\omega - 1)}{6\alpha}$
Z	$ Z(\omega) \cdot \cos\varphi(\omega) = \left -\frac{A_T(j\omega)}{B_T(j\omega)} \right \cdot \cos \left(\arg \left(-\frac{A_T(j\omega)}{B_T(j\omega)} \right) \right) =$ $= \left -\lambda \cdot \sqrt{\frac{j\omega}{\alpha}} \cdot \coth \left(L \cdot \sqrt{\frac{j\omega}{\alpha}} \right) \right \cdot \cos \left(\arg \left[-\lambda \cdot \sqrt{\frac{j\omega}{\alpha}} \cdot \coth \left(L \cdot \sqrt{\frac{j\omega}{\alpha}} \right) \right] \right)$	<p>At high frequencies</p> $ Z(\omega) \cdot \cos\varphi(\omega) \approx -\frac{\lambda}{\sqrt{2\alpha}} \cdot \omega^{0.5}$ <p>At low frequencies</p> $ Z(\omega) \cdot \cos\varphi(\omega) \approx -\frac{\lambda}{L} - \frac{\lambda \cdot L^3 \cdot \omega \cdot (e^\omega - 1)}{45 \cdot \alpha^2}$
	$ Z(\omega) \cdot \sin\varphi(\omega) = \left -\frac{A_T(j\omega)}{B_T(j\omega)} \right \cdot \sin \left(\arg \left(-\frac{A_T(j\omega)}{B_T(j\omega)} \right) \right) =$ $= \left -\lambda \cdot \sqrt{\frac{j\omega}{\alpha}} \cdot \coth \left(L \cdot \sqrt{\frac{j\omega}{\alpha}} \right) \right \cdot \sin \left(\arg \left[-\lambda \cdot \sqrt{\frac{j\omega}{\alpha}} \cdot \coth \left(L \cdot \sqrt{\frac{j\omega}{\alpha}} \right) \right] \right)$	<p>At high frequencies</p> $ Z(\omega) \cdot \sin\varphi(\omega) \approx -\frac{\lambda}{\sqrt{2\alpha}} \cdot \omega^{0.5}$ <p>At low frequencies</p> $ Z(\omega) \cdot \sin\varphi(\omega) \approx -\frac{\lambda \cdot L \cdot (e^\omega - 1)}{3\alpha}$

382

383 When the construction is made from more than one layer, approximate functions are similar to the ones
384 already shown in Table 3. They can be expressed, as well, in terms of several constants that depend on
385 the properties of the layers. However, it is more practical to estimate these constants by evaluating the
386 Modified Frequency Functions at the ends of the frequency vector used for spline interpolation; that is to
387 say, at ω_1 for the asymptotic heads and ω_2 for the asymptotic tails.

388 A particular case arises again when the innermost or the outermost layer in the construction has zero
389 density or zero thermal capacity, which is a reasonable approximation when thermal diffusivity is high
390 enough. This assumption implies a change in the asymptotic functions, as shown in Table 4.

391 Table 4: Expression of the asymptotic equivalents for a multilayer construction considering zero-inertia cases

Functions for response	Approximate functions
------------------------	-----------------------

factor calculation		High frequency			Low frequency
		Outermost layer without thermal inertia	Innermost layer without thermal inertia	All layers have thermal inertia	
X	$ X(\omega) \cdot \cos(\arg(X(\omega)))$	$\frac{\lambda_1}{L_1} - k_{XTC2} \cdot \omega^{-0.5}$ $(k_{XTC1} = \frac{\lambda_1}{L_1})$	$k_{XTC2} \cdot \omega^{0.5}$ $(k_{XTC1} = 0)$	$k_{XTC2} \cdot \omega^{0.5}$ $(k_{XTC1} = 0)$	$\frac{1}{\sum_{i=1}^n \frac{L_i}{\lambda_i}} + \lambda_{XHC2} \cdot \omega \cdot (e^\omega - 1)$ $(k_{XHC1} = \frac{1}{\sum_{i=1}^n \frac{L_i}{\lambda_i}})$
	$ X(\omega) \cdot \sin(\arg(X(\omega)))$	$k_{XTS2} \cdot \omega^{-0.5}$ $(k_{XTS1} = 0)$	$k_{XTS2} \cdot \omega^{0.5}$ $(k_{XTS1} = 0)$	$k_{XTS2} \cdot \omega^{0.5}$ $(k_{XTS1} = 0)$	$k_{XHS2} \cdot (e^\omega - 1)$ $(k_{XHS1} = 0)$
Y	$ Y(\omega) \cdot \cos(\arg(Y(\omega)))$	0	0	0	$\frac{1}{\sum_{i=1}^n \frac{L_i}{\lambda_i}} + k_{YHC2} \cdot \omega \cdot (e^\omega - 1)$ $(k_{YHC1} = \frac{1}{\sum_{i=1}^n \frac{L_i}{\lambda_i}})$
	$ Y(\omega) \cdot \sin(\arg(Y(\omega)))$	0	0	0	$k_{YHS2} \cdot (e^\omega - 1)$ $(k_{YHS1} = 0)$
Z	$ Z(\omega) \cdot \cos(\arg(Z(\omega)))$	$k_{ZTC2} \cdot \omega^{0.5}$ $(k_{ZTC1} = 0)$	$\frac{\lambda_N}{L_N} - k_{ZTC2} \cdot \omega^{-0.5}$ $(k_{ZTC1} = \frac{\lambda_N}{L_N})$	$k_{ZTC2} \cdot \omega^{0.5}$ $(k_{ZTC1} = 0)$	$-\frac{1}{\sum_{i=1}^n \frac{L_i}{\lambda_i}} + k_{ZHC2} \cdot \omega \cdot (e^\omega - 1)$ $(k_{ZHC1} = -\frac{1}{\sum_{i=1}^n \frac{L_i}{\lambda_i}})$
	$ Z(\omega) \cdot \sin(\arg(Z(\omega)))$	$k_{ZTS2} \cdot \omega^{0.5}$ $(k_{ZTS1} = 0)$	$k_{ZTS2} \cdot \omega^{-0.5}$ $(k_{ZTS1} = 0)$	$k_{ZTS2} \cdot \omega^{0.5}$ $(k_{ZTS1} = 0)$	$k_{ZHS2} \cdot (e^\omega - 1)$ $(k_{ZHS1} = 0)$

392

393 Once the constants k_{XTC1} , k_{XTC2} , k_{XTS2} , k_{XHC1} , k_{XHC2} , k_{XHS2} , k_{YHC1} , k_{YHC2} , k_{YHS2} , k_{ZTC1} , k_{ZTC2} , k_{ZTS2} , k_{ZHC1} ,
394 k_{ZHC2} and k_{ZHS2} are calculated, an accurate mathematical characterization of the MFFs for both low and
395 high frequencies is completed, which combined with the spline model for intermediate frequencies yields
396 a good approximation for the dynamics of the construction.

397

398 **7. Complexity considerations and validation analyses**

399 **7.1. Algorithm complexity**

400 The FDSI algorithm has a linear-time complexity. This means that increasing the number of response
401 factors to be calculated (for instance in order to get accurate results with improved time resolution),
402 multiplies the number of operations by the same scale factor.

403 This particular feature makes the FDSI method extremely fast compared to most of the previous
404 alternatives. There is no need to solve systems of linear equations, to apply iterative pole finding or to use
405 finite element algorithms, what would involve quadratic-time or even greater complexity and make the
406 method much slower.

407 7.2. Validations

408 For the purpose of validation, the present FDSI method was tested in two different case studies, and the
409 obtained results were compared with published data derived from previous existing methods. These case
410 studies consist of two multilayered walls of different thermal inertia that have already served as test cases
411 for other authors. Both are described in detail below.

412 Moreover, it should be noted that different criteria have been used for the comparison of the obtained
413 results. In first place stationary thermal transmittance (U-value) can be used as a checking parameter in
414 the calculation of thermal response factors, as the sum of the infinite series of response factors for a
415 given construction has to be equal to its U-value. Therefore, an error estimate can be expressed through
416 Eq.27.

$$417 E_U(\%) = \left| \frac{U - \sum_{i=1}^{\infty} RF_i}{U} \right| \cdot 100 \quad Eq. 27$$

418 However, obtaining low values for this estimate is a necessary but not sufficient condition. That is to say,
419 it does not guarantee itself the accuracy of the method, but having high E_U values always involves lack of
420 it. Then, this estimate is presented for the studied test cases as a first premise, but also further validations
421 are provided.

422 7.2.1. Case study I

423 The physical characteristics of the first test wall are presented in Table 5. It was chosen by Ouyang and
424 Haghghat [17] to demonstrate the application of their SSM approach and then, it has been used by other
425 authors to compare their own results, which derive from 3 other different methods, namely, DRF [35],
426 FDR [3] and DNI [15].

427 Table 5: Detailed wall description of case study I

Description	L (mm)	λ ($W \cdot m^{-1} \cdot K^{-1}$)	ρ ($kg \cdot m^{-3}$)	c_p ($J \cdot kg^{-1} \cdot K^{-1}$)	R ($m^2 \cdot K \cdot W^{-1}$)
Outside surface film					0.0500
Concrete	89	1.73	2235	1106	0.0514
Insulation	127	0.0744	24	992	1.7070
Concrete	89	1.73	2235	1106	0.0514
Inside surface film					0.16

428

429 Table 6 compiles the first 20 cross (Y) response factors obtained by those methods (these data are
430 extracted from [15]) and adds the corresponding results from the present FDSI approach. It can be
431 observed that the accuracy of the FDSI results is comparable to that of the methodologies used so far.

432 Particularly, considering the reference of the FDR method as the most exact among them [15], the FDSI
 433 and the DNI alternatives clearly represent better approximations. Indeed, for this case study, it is
 434 noticeable that all the calculated response factors are identical for the FDSI and the DNI methods (at
 435 least considering the precision used in Table 6), when, however, they are based on completely different
 436 conceptual approaches.

437 Table 6: Comparison of thermal response factors for case study I obtained with different calculation methods

	FDR	SSM	DRF	DNI	FDSI
0	0.00001521	0.00001771	0.00001549	0.00001531	0.00001531
1	0.00163441	0.00164078	0.00164541	0.00163463	0.00163463
2	0.00849218	0.00852682	0.00852884	0.00849216	0.00849216
3	0.01600825	0.01606351	0.01605804	0.01600833	0.01600833
4	0.02127237	0.02132861	0.02132482	0.02127245	0.02127245
5	0.02453370	0.02458189	0.02458376	0.02453375	0.02453375
6	0.02630043	0.02634117	0.02634535	0.02630044	0.02630044
7	0.02697839	0.02701426	0.02701681	0.02697837	0.02697837
8	0.02687682	0.02690951	0.02690827	0.02687681	0.02687681
9	0.02622975	0.02625774	0.02625429	0.02622975	0.02622975
10	0.02521328	0.02523350	0.02523131	0.02521329	0.02521329
11	0.02395904	0.02397017	0.02397118	0.02395907	0.02395907
12	0.02256462	0.02256861	0.02257155	0.02256466	0.02256466
13	0.02110158	0.02110207	0.02110402	0.02110163	0.02110163
14	0.01962166	0.01962103	0.01962030	0.01962172	0.01962172
15	0.01816159	0.01815949	0.01815708	0.01816165	0.01816165
16	0.01674674	0.01674130	0.01673967	0.01674679	0.01674679
17	0.01539396	0.01538425	0.01538486	0.01539401	0.01539401
18	0.01411377	0.01410104	0.01410310	0.01411382	0.01411382
19	0.01291201	0.01289871	0.01290017	0.01291206	0.01291206

438
 439 The twenty RF from Table 6 have been shown for comparison purposes, but it can be proved that they
 440 are insufficient to provide a complete representation of the transient thermal response of the selected
 441 wall. Actually, Table 7 presents the error estimate (E_U) when considering different number of thermal
 442 response factors calculated by the FDSI method.

443 Table 7: Error estimate E_U for the wall's response factors of case studies I and II calculated by the FDSI method

	Case study I Ouyang and Haghghat's wall	Case study II Chen's heavyweight wall
--	--	--

Number of RF (N)	$\sum_{i=1}^N RF_i$	Eu (%)	$\sum_{i=1}^N RF_i$	Eu (%)
20	0,3681307	25,642%	0,1846084	76,380%
50	0,4888178	1,265%	0,5783250	26,006%
100	0,4950393	0,008%	0,7491220	4,153%
150	0,4950788	0,000%	0,7763976	0,663%
200			0,7807529	0,106%
300			0,7815595	0,003%
Actual U-value	0,4950791		0,7815806	

444

445 As expected, the error is drastically reduced when considering a higher number of response factors,
446 proving an adequate fulfilment of the U-value premise. Particularly, for Case study I, 100 calculated
447 factors provide an error lower than 0.01%.

448 7.2.2. Case study II

449 Table 8 reports the physical properties of the second test construction selected for validation. It consists
450 of a heavyweight wall widely used in China and proposed by Chen et al. [36] to demonstrate the
451 application of a verification methodology for transient heat flow calculations in multilayered walls.
452 Moreover, results derived from the DNI method [15] are also available in literature for this wall, what
453 justifies the present choice.

454

Table 8: Detailed wall description of case study II

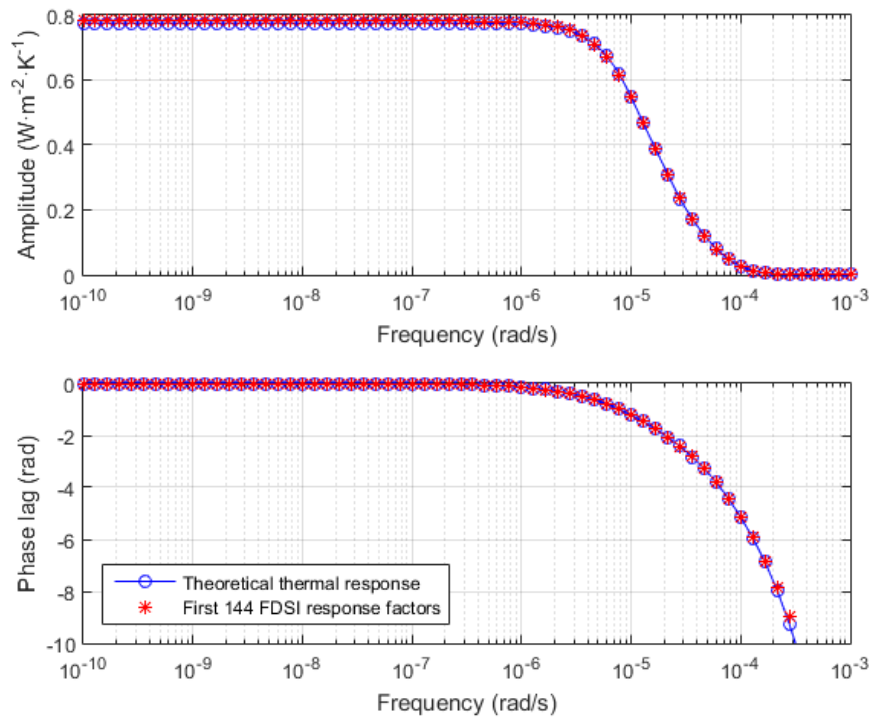
Description	L (mm)	λ (W·m ⁻¹ K ⁻¹)	ρ (kg·m ⁻³)	c_p (J·kg ⁻¹ K ⁻¹)	R (m ² ·K·W ⁻¹)
Outside surface film					0.0538
Common brick	370	0.814	1800	879	
Foam concrete	100	0.209	600	837	
Wood wool board	25	0.163	400	2093	
Stucco	20	0.814	1600	837	
Inside surface film					0.1147

455

456 In the first place, Table 7 should be referred again to check that the U-value premise is also fulfilled for
457 Case study II. Indeed, the error estimate can be reduced to negligible values (lower than 0.005 %) when
458 considering a properly high number of RF. However, in this case, approximately 300 factors are needed
459 to get the aforementioned error values. This is consistent with the heavyweight characteristics of the
460 present wall. Note that the number of response factors that are required to correctly model a given

461 construction depends on its physical properties (which determine its time constant), as well as on the
 462 considered timestep.

463 In the second place, results obtained through the FDSI method have been compared with literature
 464 values according to the verification procedure developed in [36]. This methodology proposes the
 465 comparison of the theoretical frequency characteristics of the wall (obtained from its transmission matrix)
 466 with the dynamic behavior data derived from the calculated RF. This can be done visually by representing
 467 a Bode diagram with the amplitude and phase lag of the wall's thermal response. As an example, Figure
 468 6 shows the theoretical and calculated characteristics of cross heat conduction for Case study II
 469 considering 144 factors.



470
 471 **Figure 6. Comparison between the theoretical frequency response of the heavyweight wall (case study II) and**
 472 **that obtained from the calculated response factors with the FDSI method**

473 It can be observed that both frequency characteristics are almost indistinguishable, what is also generally
 474 reported in literature for other test cases and calculation methods [3, 25, 36]. For this reason, the use of
 475 the following error estimate (Eq.28) was proposed and recommended [36].

476

$$E_{\psi} = \frac{1}{U} \sqrt{\frac{1}{N} \sum_{k=1}^N [\psi(\omega_k) - \tilde{\psi}(\omega_k)]^2} \cdot 100 \quad Eq. 28$$

477 According to this criterion, results from the FDSI, DNI and FDR methods are compared in Table 9. The
 478 first 72, 96, 120 and 144 cross (Y) factors were considered. Data from the existing methods were
 479 extracted from [15].

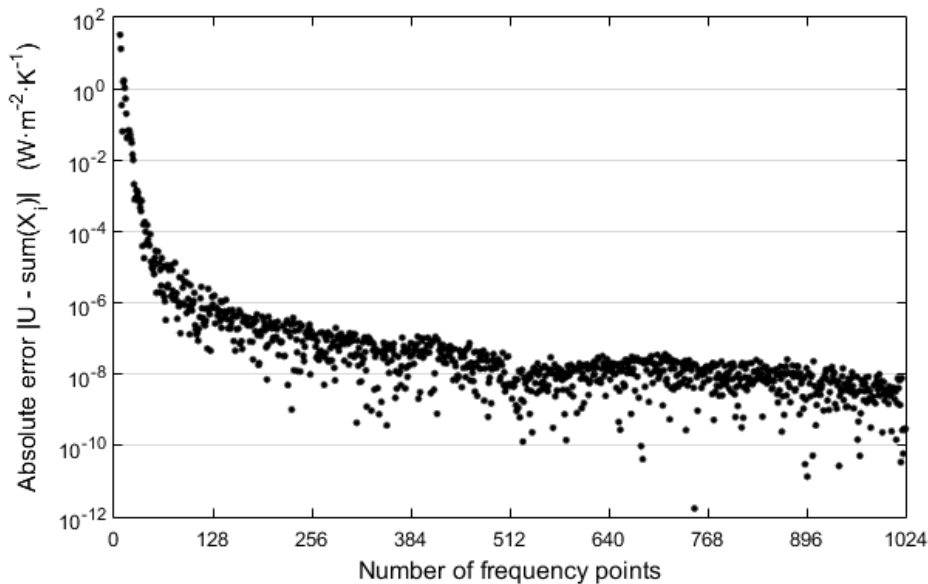
480 Table 9: Compared accuracy for FDSI, DNI and FDR methods for 'Case study II' wall.

Number of RF	72	96	120	144
E_{ψ_FDSI} (%)	9.15	3.76	1.55	0.64
E_{ψ_DNI} (%)	8.92	3.66	1.50	0.62
E_{ψ_FDR} (%)	9.12	3.74	1.54	0.63

481
 482 It can be seen that the obtained errors for the FDSI method, despite being slightly higher, are very close
 483 to those of the DNI and the FDR method. As pointed out in [15], the latter, by definition, minimizes the
 484 present error estimate E_{ψ} , so the observed small differences prove a very interesting behavior of the FDSI
 485 method.

486 7.3. Accuracy dependence on the number of frequency points

487 Figure 7 shows an estimation of the absolute difference between the actual U-value and the sum of 300
 488 calculated X response factors as a function of the number of frequency points used by the FDSI method
 489 when applied to 'Case study I'. Similar results can be obtained for Y and Z response factor series.



490
 491 **Figure 7. Influence of the number of frequency points on the method's accuracy**

492 As expected, Figure 7 reveals that the accuracy of the FDSI method improves as the number of
 493 frequency points becomes larger, because the MFFs are better estimated. However, once the
 494 mathematical approximation is good enough, there is no use in increasing the number of interpolation

495 points. With 512 frequency evaluations and a sufficient number of factors, the absolute error falls easily
496 under 10^{-8} . Better estimates could be obtained by widening the frequency range or by calculating more
497 response factors. Nevertheless, here the conceptual development and first validations of the FDSI
498 method are presented, but further work needs to be done in this sense in the future.

499 Moreover, it should be noted that data in Figure 7 come from an indirect calculation procedure that
500 provides approximated results. Only the pre-calculated integration factors for the case with 1024
501 frequency points were actually determined. For the other frequency sampling cases, the corresponding
502 pre-calculated constants required by the FDSI method were estimated by a grouping routine. This aspect
503 explains the scattering of the plot shown in Figure 7 and suggests not considering those exceptional
504 accuracy values associated to certain singular frequencies.

505 **8. Conclusions**

506 This work has introduced a new method for the calculation of conduction response factors in multilayer
507 constructions, based on frequency-domain spline interpolation (FDSI) and asymptotic analysis. Its
508 conceptual development as well as first validations comparing with existing methods from previous
509 literature have been presented.

510 The FDSI method enables the calculation of thermal response factors with great accuracy and speed,
511 which constitutes a promising alternative to improve those procedures implemented in Building Energy
512 Simulation programs so far. Particularly, it can make BES tools able to efficiently calculate with small
513 timesteps (1-5 min) which is of special interest in energy simulations combining buildings and HVAC
514 systems that often have much shorter time responses. In order to run simulations with small timesteps
515 (keeping the level of accuracy), the only requirement is to calculate more response factors, but thanks to
516 the lookup table approach most of the involved calculations will be done only once and stored, so the
517 'small timestep' condition will not increase a lot the computational effort and make such simulations
518 affordable.

519 In summary, the following features characterize the new proposed method:

- 520 • Precision: As integration factors can be pre-calculated beforehand with excellent precision, error
521 depends only on the number of frequency evaluations, the number of factors and the width of the
522 frequency range. With 512 frequency evaluations and a sufficient number of factors, the estimate for
523 this variable falls easily under 10^{-8} .

524 • Speed: This algorithm has linear-time complexity, which makes it extremely fast compared to other
525 methods. There is no need to solve linear systems, iterative pole finding or the use of finite element
526 algorithms. It just requires N frequency evaluations, the calculation of spline coefficients and $8 \cdot N$
527 multiplications for each response factor.

528 • Stability: FDSI method is inherently stable. Convergence is always guaranteed because there are no
529 iterative numeric algorithms involved.

530 As a drawback, this method requires several megabyte of RAM to store the pre-calculated integration
531 factors. However, since modern computers are able to handle up to several gigabytes of main memory,
532 this is not a relevant issue.

533

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538

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- 624

626 Table 1: Summary of different coefficients relevant for the FDSI method

Type of factor		Integration factors (pre-calculated factors)		Coefficients that depend on the construction			
				MFF _x response factors	MFF _y response factors	MFF _z response factors	
For each response factor (asymptotic terms)	Asymptotic tail terms	MFF cos	X	Outermost layer without thermal inertia $KATC1_n = KATC0_n$ $KATC2_n = KATC(+)_n$	$k_{XTC1} = \frac{\lambda_1}{L_1}$ k_{XTC2}		
			Other cases $KATC1_n = 0$ $KATC2_n = KATC(+)_n$	$k_{XTC1} = 0$ k_{XTC2}			
		Y	No coefficients used (its contribution is negligible)				
		Z	Innermost layer without thermal inertia $KATC1_n = KATC0_n$ $KATC2_n = KATC(-)_n$				$k_{XTC1} = \frac{\lambda_N}{L_N}$ k_{XTC2}
			Other cases $KATC1_n = 0$ $KATC2_n = KATC(+)_n$				$k_{XTC1} = 0$ k_{XTC2}
		MFF sin	X	Outermost layer without thermal inertia $KATS1_n = 0$ $KATS2_n = KATS(-)_n$	$k_{XTS1} = 0$ k_{XTS2}		
	Other cases $KATS1_n = 0$ $KATS2_n = KATS(+)_n$			$k_{XTS1} = 0$ k_{XTS2}			
	Y		No coefficients used (its contribution is negligible)				
	Z		Innermost layer without thermal inertia $KATS1_n = 0$ $KATS2_n = KATS(-)_n$				$k_{ZTS1} = 0$ k_{ZTS2}
		Other cases $KATS1_n = 0$ $KATS2_n = KATS(+)_n$	$k_{ZTS1} = 0$ k_{ZTS2}				
Asymptotic head terms	MFF cos	$KAHC1_n$ $KAHC2_n$		$k_{XHC1} = \frac{1}{\sum_{i=1}^n \frac{L_i}{\lambda_i}}$ k_{XHC2}	$k_{YHC1} = \frac{1}{\sum_{i=1}^n \frac{L_i}{\lambda_i}}$ k_{YHC2}	$k_{ZHC1} = \frac{-1}{\sum_{i=1}^n \frac{L_i}{\lambda_i}}$ k_{ZHC2}	
	MFF sin	$KAHS1_n$ $KAHS2_n = 0$		k_{XHS1} $k_{XHS2} = 0$	k_{YHS1} $k_{YHS2} = 0$	k_{ZHS1} $k_{ZHS2} = 0$	
For each interval and response factor (polynomial terms)	Spline ω^3	MFF cos	$KPCa_{n,k}$		$a_{XC(k)}$	$a_{YC(k)}$	$a_{ZC(k)}$
		MFF sin	$KPSa_{n,k}$		$a_{XS(k)}$	$a_{YS(k)}$	$a_{ZS(k)}$
	Spline ω^2	MFF cos	$KPCb_{n,k}$		$b_{XC(k)}$	$b_{YC(k)}$	$b_{ZC(k)}$
		MFF sin	$KPSb_{n,k}$		$b_{XS(k)}$	$b_{YS(k)}$	$b_{ZS(k)}$
	Spline ω	MFF cos	$KPCc_{n,k}$		$c_{XC(k)}$	$c_{YC(k)}$	$c_{ZC(k)}$
		MFF sin	$KPSc_{n,k}$		$c_{XS(k)}$	$c_{YS(k)}$	$c_{ZS(k)}$
	Spline ω^0	MFF cos	$KPCd_{n,k}$		$d_{XC(k)}$	$d_{YC(k)}$	$d_{ZC(k)}$
		MFF sin	$KPSd_{n,k}$		$d_{XS(k)}$	$d_{YS(k)}$	$d_{ZS(k)}$

627

628 **Appendix B. Calculation of the spline coefficients**

629 The coefficients of each cubic polynomial for the MFFs spline interpolation can be easily expressed in
 630 terms of the evaluated frequencies (ω_k), the MFF values at those frequencies (y_k) and the corresponding
 631 second derivatives ($y_k'' = \sigma_k$), as it is shown in Eqs. B.5.

632 As ω_k and y_k are known, the proposed calculation of the spline coefficients is focused on determining the
 633 second derivatives. This can be done through the recursive procedure described in Table B.1. For the
 634 sake of clarity, the following quantities have been defined:

635
$$h_k = \omega_{k+1} - \omega_k \quad 0 < k < n \quad \text{Eq. B.1a}$$

636
$$r_k = \frac{y_{k+1} - y_k}{\omega_{k+1} - \omega_k} \quad 0 < k < n \quad \text{Eq. B.1b}$$

637 Then, φ_k and Γ_k auxiliary terms are obtained in intermediate steps to finally calculate the MFF second
 638 derivatives and the spline coefficients.

639 Table B.1.: Recursive calculation procedure to get the spline coefficients in a global coordinate system

<p>1.- First, φ_k terms are calculated. It requires $n-1$ iterations, where n is the number of frequency evaluations.</p> <p>Eqs. B.2.</p> $\begin{cases} \varphi_1 = \frac{h_2 \cdot (2h_2 - h_1)}{(h_2 - h_1)} \\ \varphi_k = 2 \cdot (h_{k+1} + h_k) - \frac{h_k^2}{\varphi_{k-1}} & 1 < k < (n-1) \\ \varphi_{n-1} = \left(h_{n-2} - \frac{h_{n-1}^2}{h_{n-2}} \right) + \frac{h_{n-1}}{\varphi_{n-2}} \cdot \left(3h_{n-1} + h_{n-2} + 2 \frac{h_{n-1}^2}{h_{n-2}} \right) \end{cases}$	<p>2.- Once φ_k terms are stored, we can obtain a sequence of Γ_k terms, using the recurrence formulae shown below. Again, it requires $n-1$ iterations.</p> <p>Eqs. B.3.</p> $\begin{cases} \Gamma_1 = \frac{6h_2^2}{(h_2^2 - h_1^2)} \cdot (r_2 - r_1) \\ \Gamma_k = 6 \cdot (r_{k+1} - r_k) - \frac{h_k \cdot \Gamma_{k-1}}{\varphi_{k-1}} & 1 < k < (n-1) \\ \Gamma_{n-1} = \frac{-6h_{n-1}}{h_{n-2}} \cdot (r_{n-1} - r_{n-2}) + \frac{\Gamma_{n-2}}{\varphi_{n-2}} \left(3h_{n-1} + h_{n-2} + 2 \frac{h_{n-1}^2}{h_{n-2}} \right) \end{cases}$
<p>3.- Given φ_k and Γ_k terms, we can get the second derivatives of the splines σ_k for each evaluation point. Then, note that n σ_k-terms will be needed.</p> <p>Eqs. B.4.</p> $\begin{cases} \sigma_n = \frac{\Gamma_{n-1}}{\varphi_{n-1}} \\ \sigma_k = \frac{\Gamma_{k-1} - h_k \cdot \sigma_{k+1}}{\varphi_{k-1}} & 1 < k < n \\ \sigma_1 = \frac{\frac{-6h_1}{h_2} \cdot (r_2 - r_1) + \left(3h_1 + h_2 + 2 \frac{h_1^2}{h_2} \right) \sigma_2}{h_2 - \frac{h_1^2}{h_2}} \end{cases}$	<p>4.- Finally, the coefficients of the splines can be obtained from the list of second derivatives using the following formulae.</p> <p>Eqs. B.5.</p> $\begin{aligned} a_k &= \frac{\sigma_{k+1} - \sigma_k}{6h_k} \\ b_k &= \frac{\sigma_k \omega_{k+1} - \sigma_{k+1} \omega_k}{2h_k} \\ c_k &= \frac{\sigma_{k+1} \left(\frac{3 \cdot \omega_k^2}{h_k} - h_k \right) - \sigma_k \left(\frac{3 \cdot \omega_{k+1}^2}{h_k} - h_k \right)}{6} + \frac{y_{k+1} - y_k}{h_k} \\ d_k &= \frac{\sigma_k \left(\frac{\omega_{k+1}^3}{h_k} - h_k \omega_{k+1} \right) - \sigma_{k+1} \left(\frac{\omega_k^3}{h_k} - h_k \omega_k \right)}{6} - \frac{y_{k+1} \omega_k - y_k \omega_{k+1}}{h_k} \end{aligned}$ <p>$0 < k < n$</p>